

Metal Structures

Design Project II

Floor girders – examples of calculation (part III)

IVth example - supports of primary beam



Photo: homeownercosts.co.uk



Photo: Author

Primary beam is supported in three points:

- left end – masonry wall;
- half of span – steel column;
- right end – masonry wall;

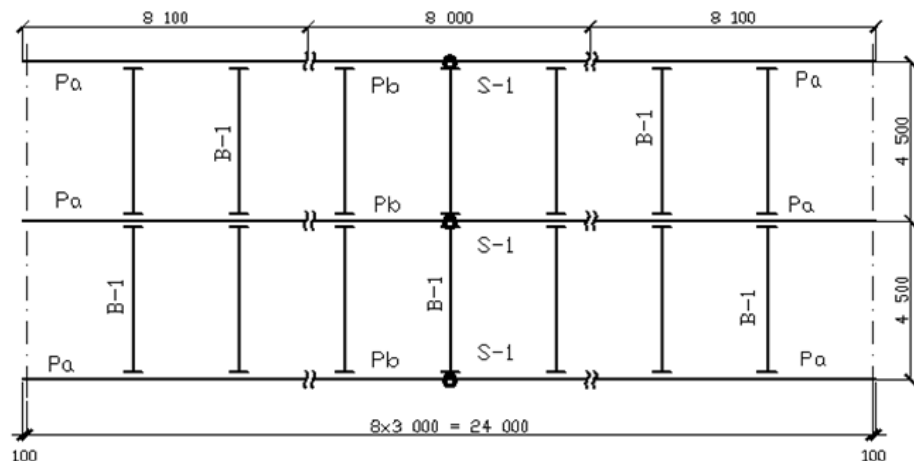
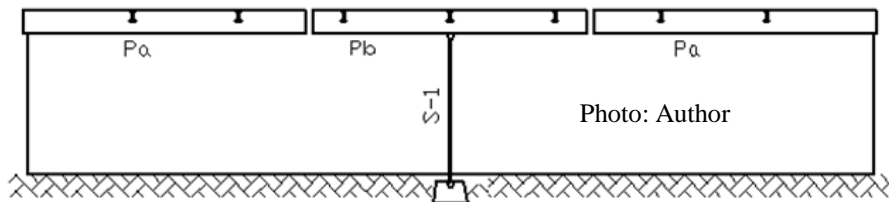


Photo: Author

New complex of data:

S235

Primary beam HEA 700

Span 2 x 12,00 m

Column HEA 240

Height 4,0 m

Reactions:

1 184,235 kN central support

391,112 kN side supports

Masonry wall:

total thickness 40 cm

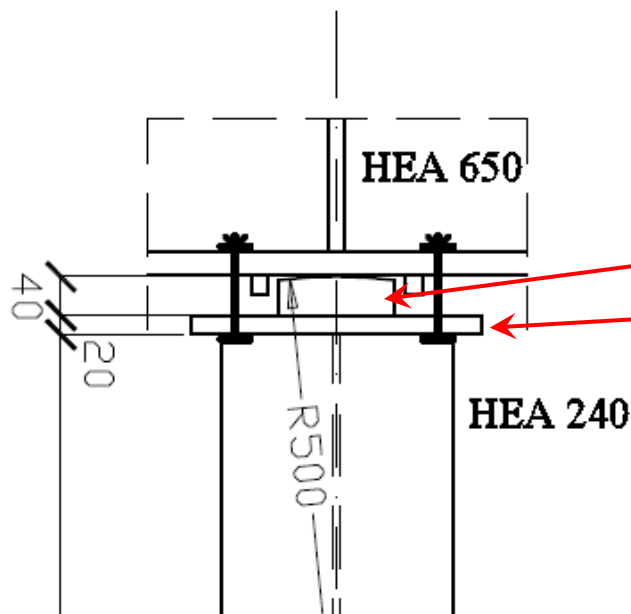
$f_d = 4,0$ MPa

$E_d = 3,6$ GPa

Analysis of supporting joint concerns few important questions:

- Initial analysis (category of joint, class of bolt, dimension, length, geometry) → #t / 5 - 6
- Stiffness of joint (according assumption – hinge joint) → #t / 7 - 13
- Resistance of rockers → #t / 14 - 19
- Resistance of masonry wall → #t / 20 - 22
- Resistance of head of column → #t / 23 - 30
- Stiffener → **example Vth**

Initial assumptions



Support consists on
rocker
bottom plate.

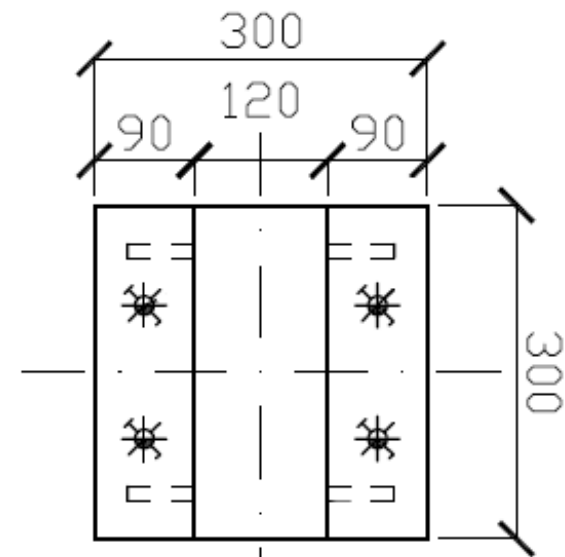
Bottom plate is in contact with:

masonry wall → #t / 20 - 22

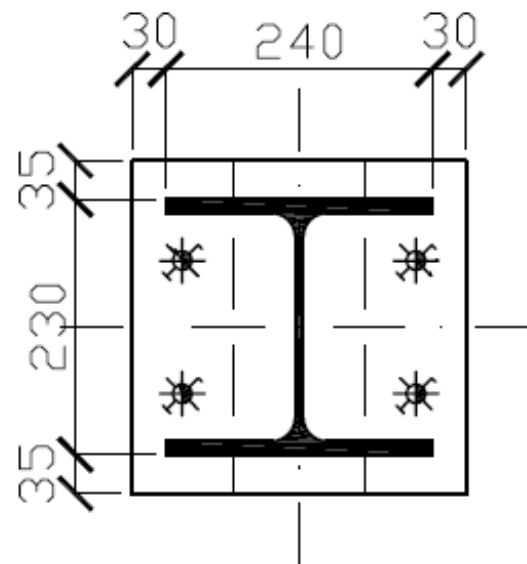
or

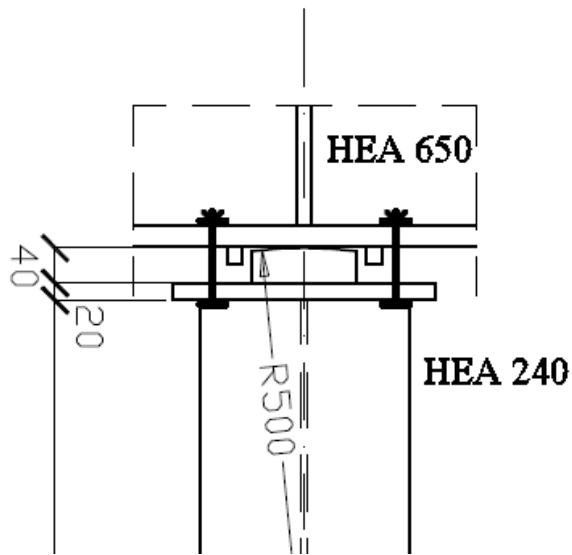
steel column → #t / 23 - 30.

The same geometry of support will be adopted in both cases. Bigger force (reaction on side support / central support) will be applied in calculations.

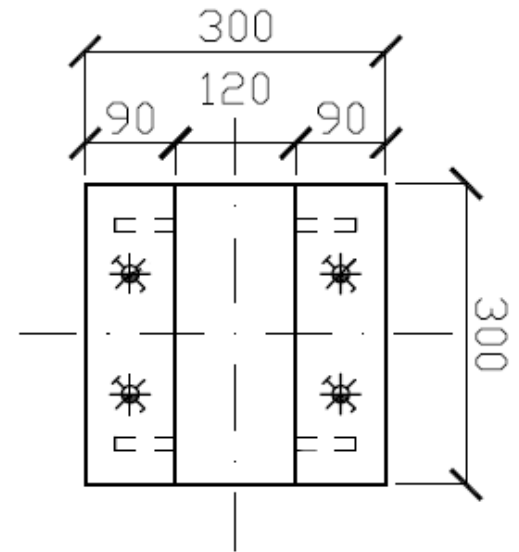


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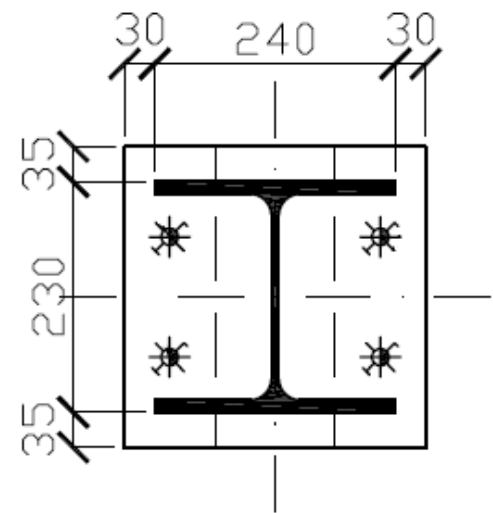
Thickness of bottom plate similar to thickness of bottom flange of primary beam; adopted 20 mm

Thickness of rocker about 2x bigger than thickness of bottom plate; adopted 40 mm

Radius of round part (if is neccessible) about 10x bigger than rocker thickness; adopted 500 mm

Bottom plate dimnesnin: bigger than envelope of column cross-section, but not greated than masonry wall thickness; adopted 300x300 mm

Rocker width about 1/3 bottom plate width; adopted 120 mm



Stiffness

For „normal” truss and beams important factor is proportion between stiffness of Girder (truss or I-beam) and Support (column, wall...):

$$S_{\text{Girder}} / S_{\text{Support}}$$

$$S_{\text{Girder}} = E_G J_G / L_G$$

$$S_{\text{Support}} = E_S J_S / h_S$$

Stiffness of truss could be approximated by:

$$S_{\text{truss}} \approx 3,5 q L_G^2 \quad ; \quad q = (\Sigma F_i) / L_G \quad \rightarrow \#14 / 29$$

or (\rightarrow #9 / 11):

$$J_{\text{truss}} \approx 0,35 h_{1, \text{truss}}^2 A_{\text{chord}}$$

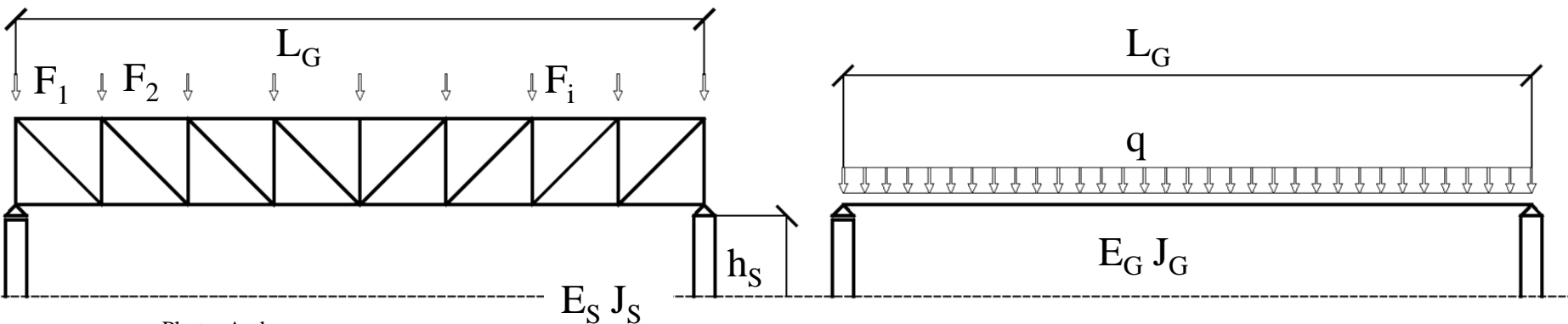


Photo: Author

Recommended technical solutions (based on experience):

(Big)	Various proportion S_G / S_S		(Small)
No rocker	Flat rocker	Round rocker	Elastomeric rocker

Elastomeric rocker is used, first of all, for bridges. For other type of structures is used very rare.

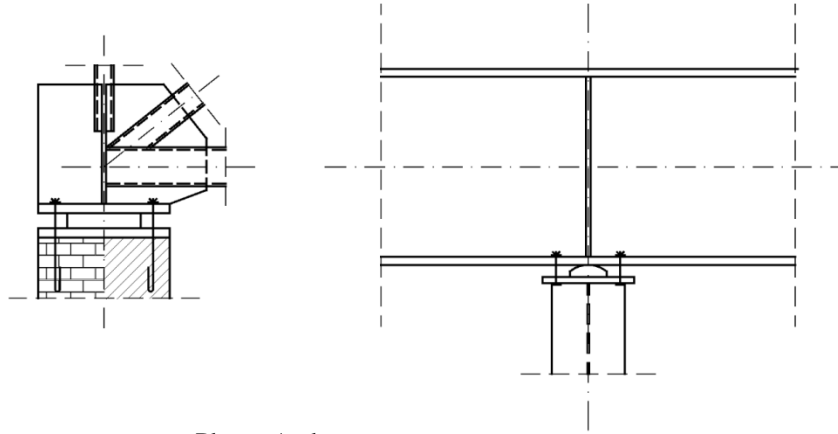


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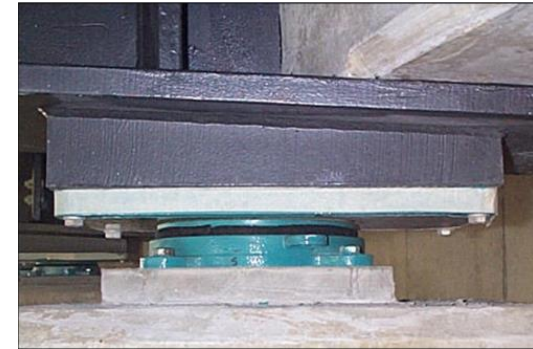


Photo: steelconstruction.info

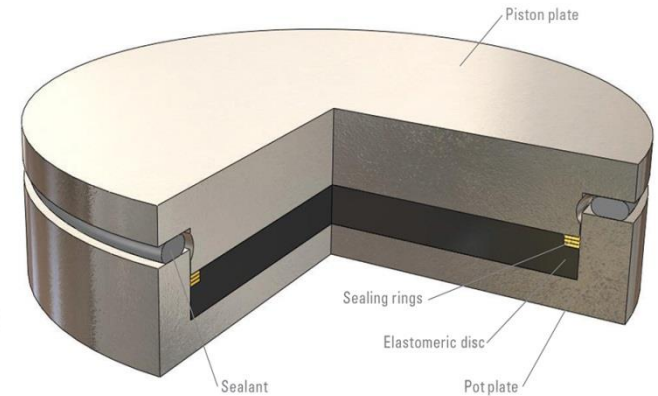


Photo: canambridges.com

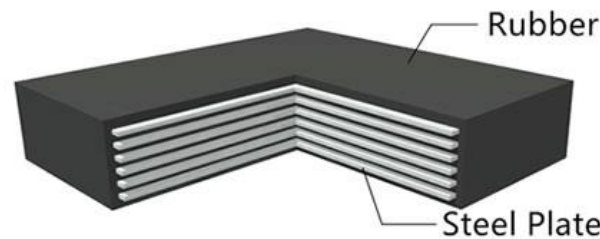


Photo: bridgebearing.org



Photo: anthony-johnson-engineering.co.uk

→ #14 / 30

Recommendations for CC3 are not necessarily valid for lower Consequence Classes. Structure should be as cheap as possible. Detailed design of supports can be very important for CC3. For lower classes, with a smaller assumed safety margin, it may not be necessary: too expensive in relation to safety of structure. Small differences between ideal and real behavior of structure can be accepted. Carrying out / resigning from thorough analysis should result from experience of designer.

CC	$S_G / S_S > 20$	$20 \geq S_G / S_S > 10$	$10 \geq S_G / S_S > 5$	$5 \geq S_G / S_S$
3	No rocker	Flat rocker	Round rocker	Elastomeric rocker
2	No rocker	No rocker / flat rocker	Flat rocker / round rocker	Round rocker
1	There is usually no detailed analysis			

→ #14 / 33

$$L = 2 L_{\text{span}} = 24,0 \text{ m}$$

$$h \text{ (HEA 700)} = 690 \text{ mm}$$

$$d = \max (h / 10 ; D_w / 2) = \\ = \max (69 \text{ mm} ; 200 \text{ mm}) = 200 \text{ mm}$$

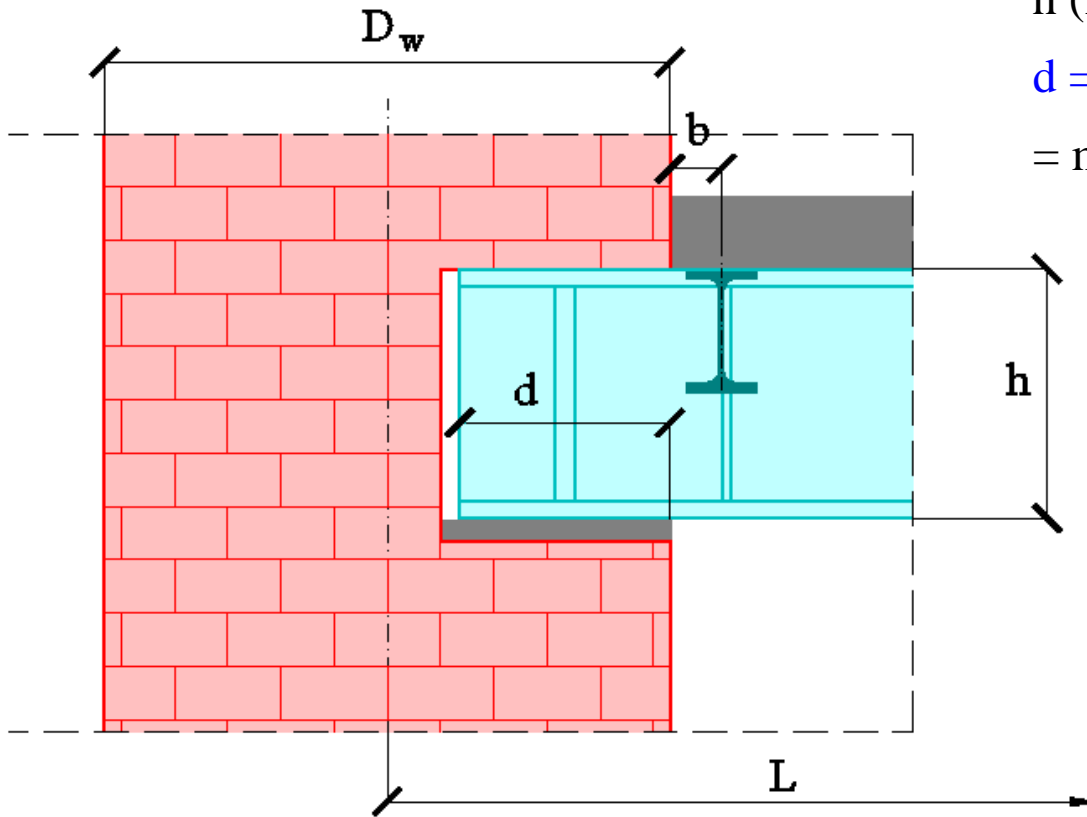


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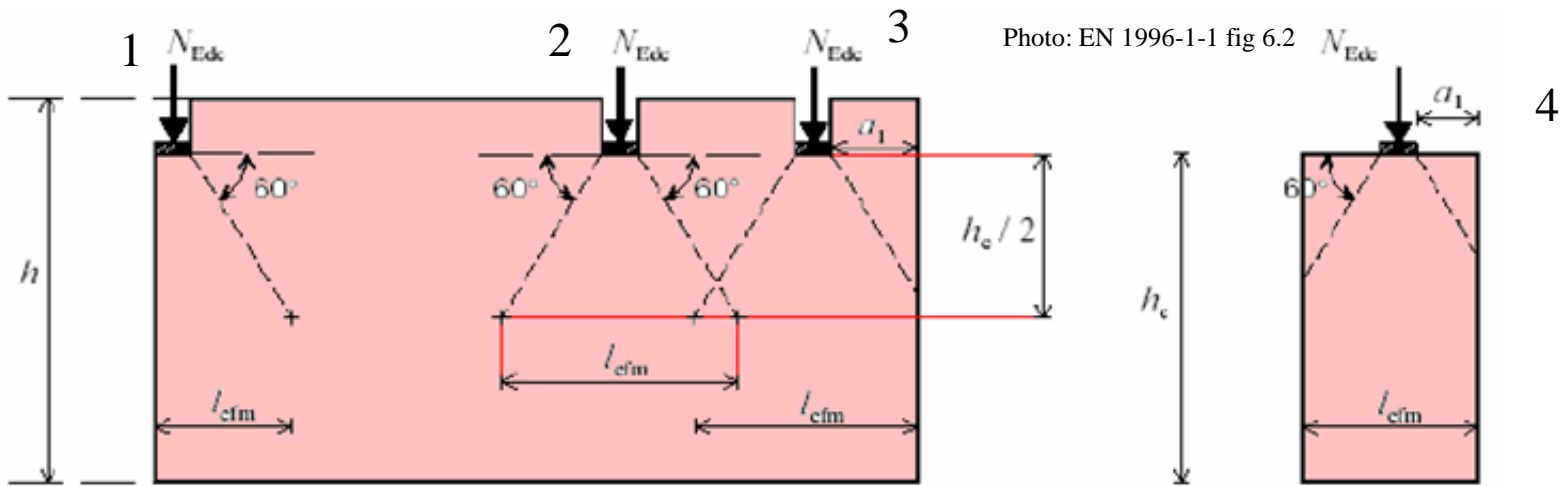


Photo: EN 1996-1-1 fig 6.2

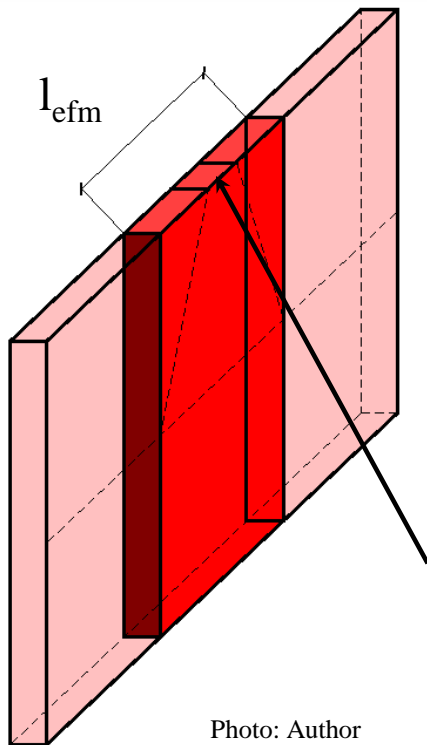
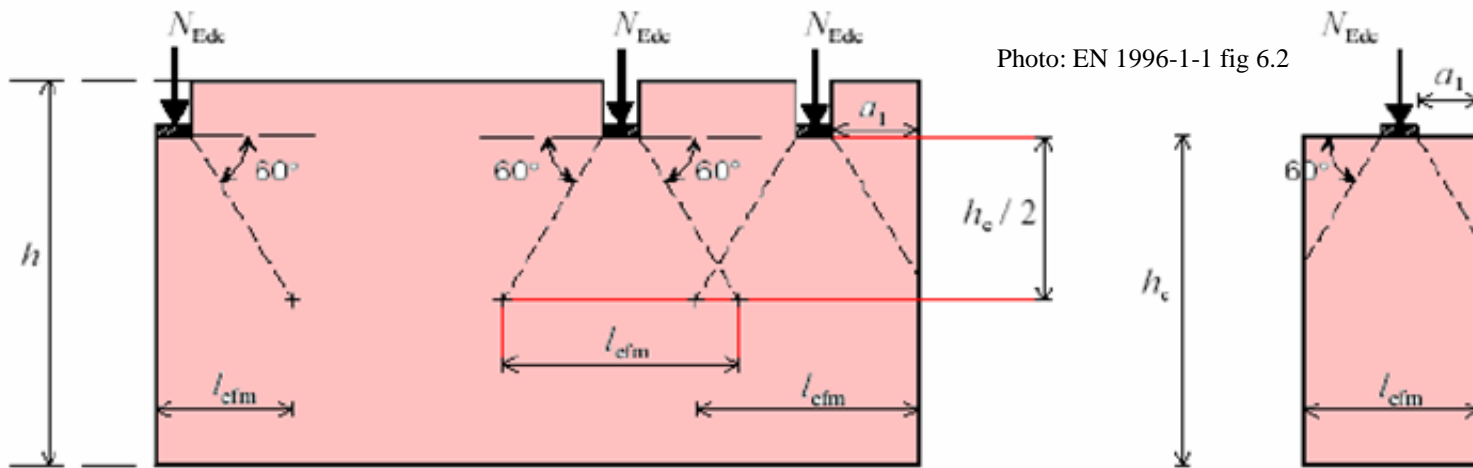


Photo: projektonwestor.pl



Photo: houzz.co.nz

Case 2 will be taken into consideration in analysis; primary beams in central part of long masonry wall.



$$h_c = 4,5 \text{ m}$$

$$l_{efm} = b(\text{HEA } 700) + 2 \times (h_c / 2) / \text{tg } 60^\circ = 30 \text{ cm} + 259,8 \text{ cm} = 2,989 \text{ m}$$

$$A = 40 \text{ cm} \times 259,8 \text{ cm} = 10\,392,3 \text{ cm}^2$$

$$J = [(40 \text{ cm})^3 \times 259,8 \text{ cm}] / 12 = 1\,385\,600 \text{ cm}^4$$

$$A_b = 20 \text{ cm} \times 30 \text{ cm} = 600 \text{ cm}^2$$

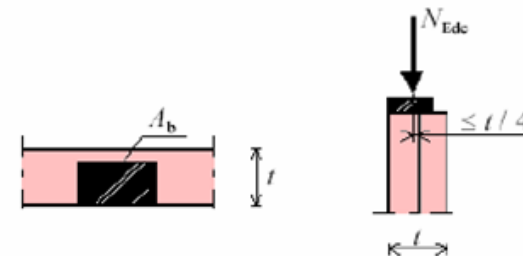


Photo: EN 1996-1-1 fig 6.2

The most popular, CC2, will be analysed.

Masonry wall:

$$E_{mw} = 3,6 \text{ GPa}$$

$$H_{mw} = 4,50 \text{ m}$$

$$J_{mw} = 1\,385\,600 \text{ cm}^4 = 0,013856 \text{ m}^4$$

$$S_{mw} = 11\,084,8 \text{ kN m}$$

$$S_{pb} / S_{mw} = 1,700 \rightarrow \text{round rocker}$$

Primary beam (HEA 700):

$$E_{pb} = 210,0 \text{ GPa}$$

$$L_{pb} = 24,0 \text{ m}$$

$$J_{pb} = 215\,300 \text{ cm}^4$$

$$S_{pb} = 18\,838,75 \text{ kN m}$$

Column (HEA 240):

$$E_c = 210,0 \text{ GPa}$$

$$H_c = 4,0 \text{ m}$$

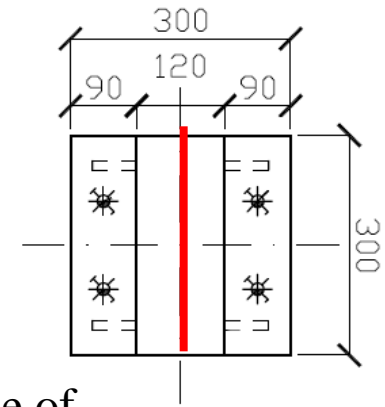
$$J_c = 7\,763 \text{ cm}^4$$

$$S_c = 4\,075,58 \text{ kN m}$$

$$S_{pb} / S_c = 4,622 \rightarrow \text{round rocker}$$

Rocker resistance

Photo: Author



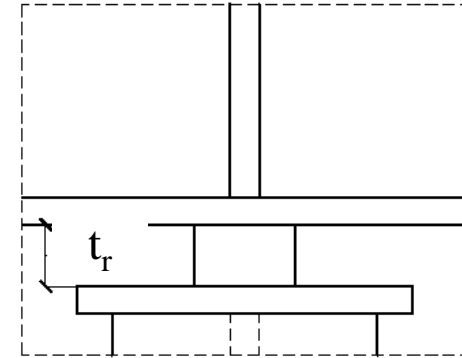
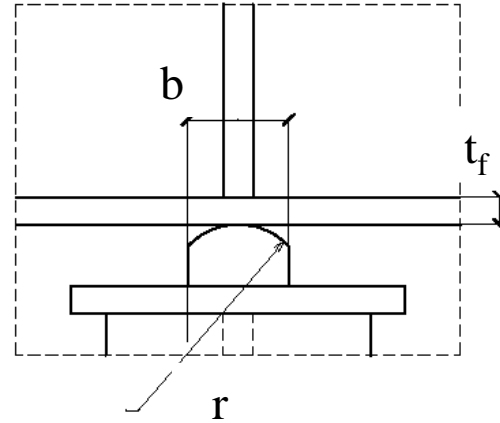
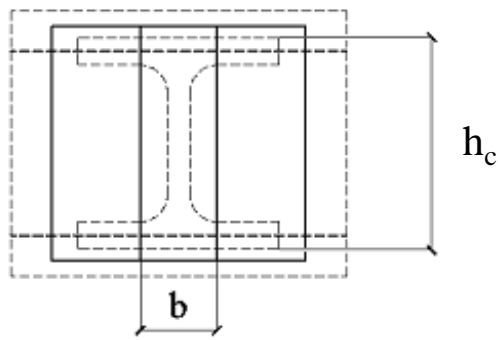
Four points are important for checking resistance:

- behavior of flat surface of beam's flange in contact with round surface of rocker (beam's resistance);
- behavior of round surface of rocker in contact with flat surface of beam's flange (rounded plate's resistance);
- contact two flat surfaces (rocker – bottom plate);
- welds between rocker and bottom plate;

In both cases (rocker on masonry wall and rocker on column's top) contact between flange and rounded plate is the same: across flange, on distance 300 mm. This is linear contact; theoretical width of it is equal 0.

Geometry the same for both cases; more important will be case with bigger force (column's rocker).

$$L \approx b_f \approx h_c$$



Rocker-beam compression

EN 1337

Condition for beam (flange resistance):

$$[N_{Ed} / b_f] / [23 r f_u^2 / (E \gamma_M)] \leq 1,0 \quad \gamma_M = 1,0$$

Condition for rounded plate:

$$[N_{Ed} / L] / [f_y (2 t_r + b) / \gamma_M] \leq 1,0 \quad \gamma_M = 1,1$$

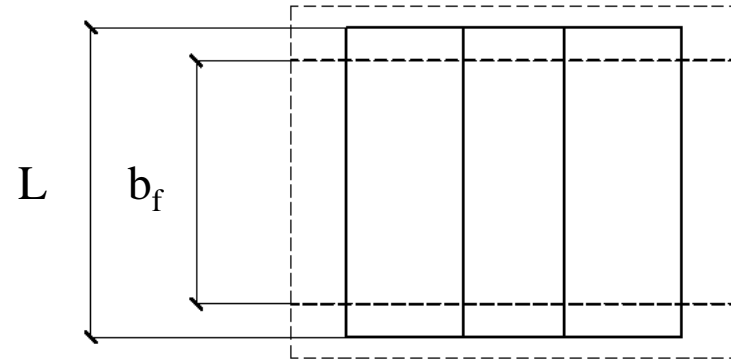


Photo: Author

$$t_r = 40 \text{ mm}$$

$$b = 120 \text{ mm}$$

$$L = 300 \text{ mm}$$

$$r = 500 \text{ mm}$$

S235

$$f_y = 235 \text{ MPa}$$

$$f_u = 360 \text{ MPa}$$

Condition for beam:

$$[N_{Ed} / b_f] / [23 r f_u^2 / (E \gamma_M)] \leq 1,0$$

$$[1\ 184,235\ \text{kN} / 0,3\ \text{m}] / [23 \cdot 0,5\ \text{m} \cdot (360\ \text{MPa})^2 / (210\ \text{GPa} \cdot 1,0)] =$$
$$= [3\ 947,450\ \text{kN/m}] / [7\ 079,143\ \text{kN/m}] = 0,558 < 1,0 \text{ OK}$$

Condition for rounded plate:

$$[N_{Ed} / L] / [f_y (2 t_r + b) / \gamma_M] \leq 1,0$$

$$[1\ 184,235\ \text{kN} / 0,3\ \text{m}] / [235\ \text{MPa} \cdot (2 \cdot 0,04\ \text{m} + 0,3\ \text{m}) / 1,1] =$$
$$= [3\ 947,450\ \text{kN/m}] / [81\ 181,818\ \text{kN/m}] = 0,049 < 1,0 \text{ OK}$$

Rocker-cap plate weldings

Lecture #17, Example 3a
for

$$M_{Ed} = 0 \quad ; \quad V_{Ed} = 0$$

$$\sqrt{[(\sigma_{\perp})^2 + 3(\tau_{\parallel}^2 + \tau_{\perp}^2)]} \leq f_u / (\beta_w \gamma_{M2})$$

$$\sigma_{\perp} \leq 0,9 f_u / \gamma_{M2}$$

$$\gamma_{M2} = 1,25$$

t_1 = thickness of bottom plate = 20 mm

t_2 = thickness of rounded plate = 40 mm

$0,2 t_2 \leq a \leq 0,7 t_1 \rightarrow a = 10 \text{ mm}$

l = length of weld = 300 mm

$N_{Ed} = 1\,184,235 \text{ kN}$

$f_u = \text{S235} = 360 \text{ MPa}$

$\beta_w = \text{S235} = 0,8$

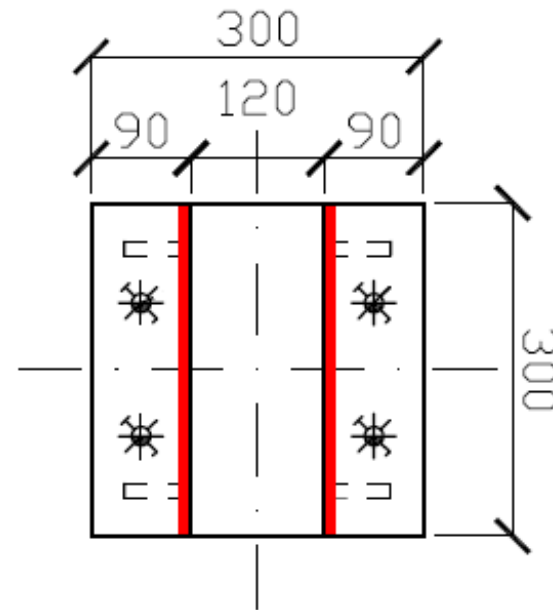


Photo: Author

$$A = 2 \text{ a } l = 60 \text{ cm}^2$$

$$\sigma (N_{Ed}) = N_{Ed} / A = 197,373 \text{ MPa}$$

$$\sigma_{\perp} = \tau_{\perp} = \sigma / \sqrt{2} = 139,565 \text{ MPa}$$

$$\tau_{\parallel} = 0,000 \text{ MPa}$$

$$\sqrt{[(\sigma_{\perp})^2 + 3(\tau_{\parallel}^2 + \tau_{\perp}^2)]} = 279,126 \text{ MPa}$$

$$f_u / (\beta_w \gamma_{M2}) = 360,0 \text{ MPa}$$

$$279,126 / 360,0 = 0,775$$

condition satisfied

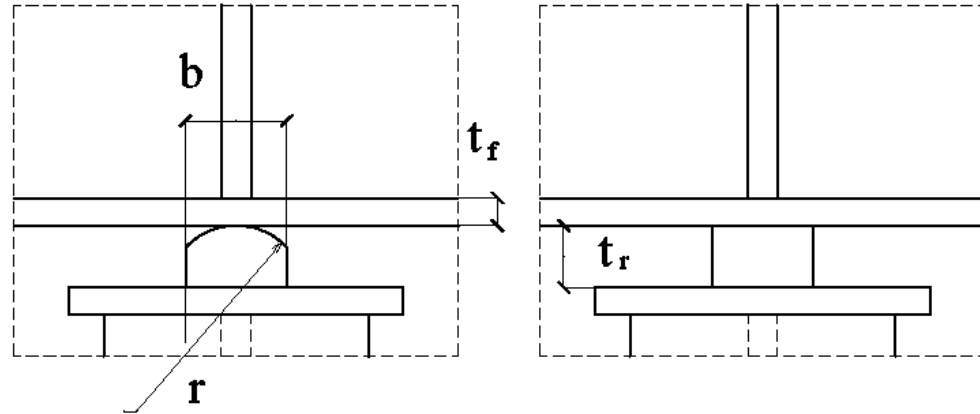
$$\sigma_{\perp} = 139,565 \text{ MPa}$$

$$0,9 f_u / \gamma_{M2} = 259,200 \text{ MPa}$$

$$139,565 / 259,200 = 0,538$$

condition satisfied

Plate – plate contact



$$N_{Ed} / (L b f_y / \gamma_{M0}) \leq 1,0$$

$$1\,184,235 \text{ kN} / (0,3 \text{ m} \cdot 0,12 \text{ m} \cdot 235 \text{ MPa} / 1,0) =$$

$$= 0,140 < 1,0 \text{ OK}$$

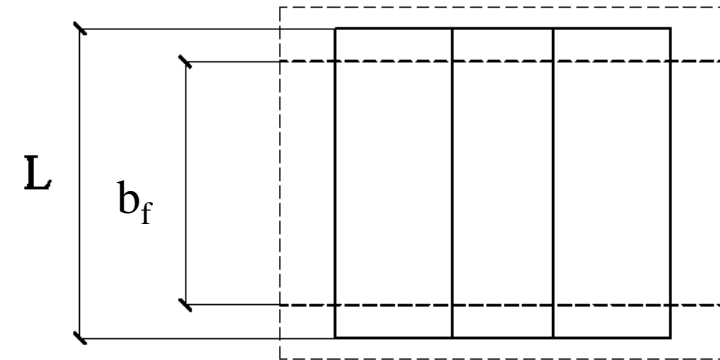
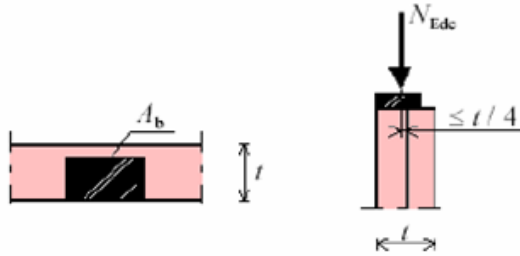


Photo: Author

Resistance of masonry wall below support



$$N_{Edc} / N_{Rdc} \leq 1,0$$

$$N_{Rdc} = b A_b f_d$$

$$A_{ef} = l_{efm} t$$

Photo: EN 1996-1-1 fig 6.2

$$b = \min \{ 1,25 + a_1 / (2 h_c) ; 1,5 ; \max [(1 + 0,3 a_1 / h_c) \cdot (1,5 - 1,1 A_b / A_{ef}) ; 0] \}$$

In case of support in central part of long masonry wall, distance a_1 from support to end of wall is very big. For sure, first and third proposal for b will be bigger than 1,5.

$$b = 1,5$$

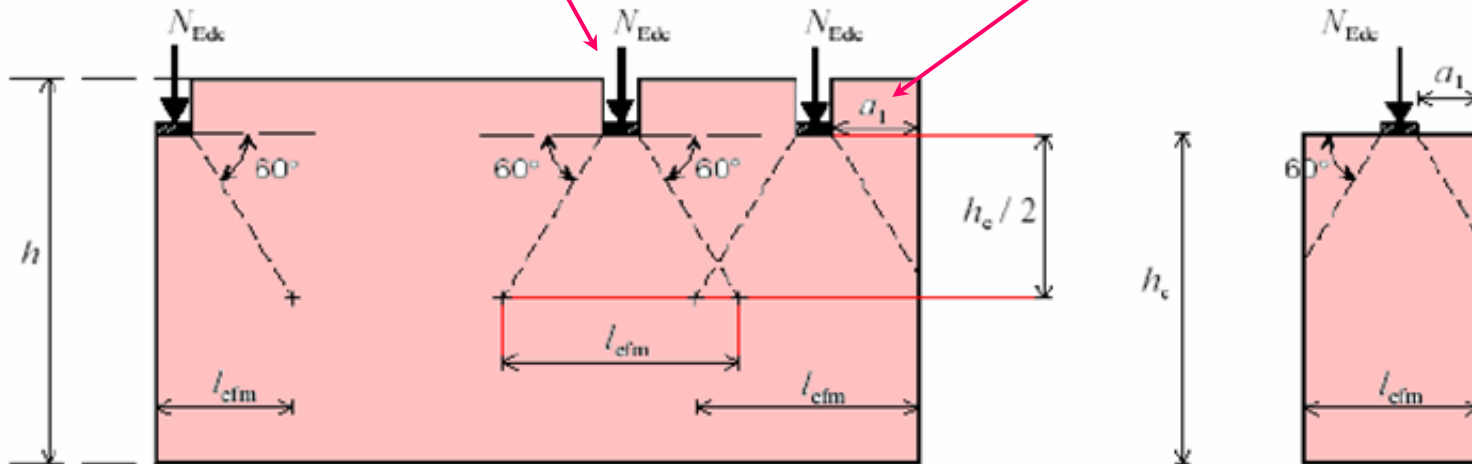


Photo: EN 1996-1-1 fig 6.2

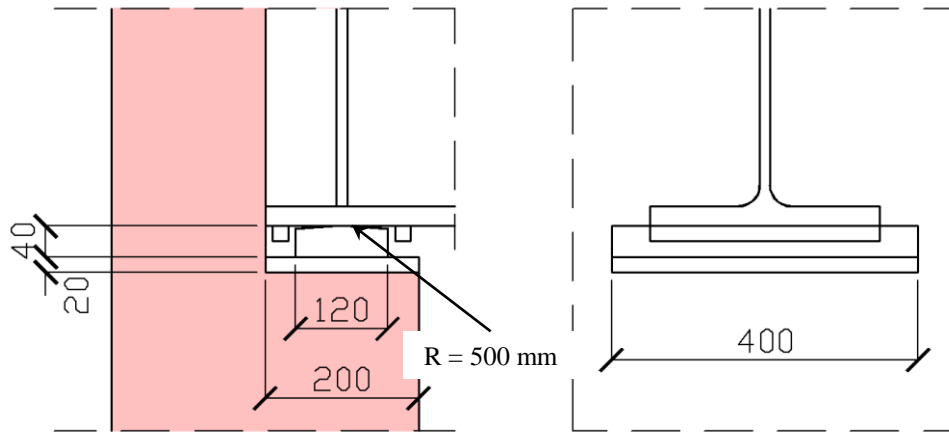


Photo: Author

At now, geometry of support on wall must be analysed once again – with additional member: rocker. Rocker is, in this case, complex of two plates: top with round up surface (common name „rocker” concerns top plate) 120x400, 40 mm thickness, $r = 500$ mm), and bottom, in contact with masonry wall (200x400, 20 mm thickness).

$$b = 1,5$$

$$A_b = 20 \text{ cm} \times 40 \text{ cm} = 800 \text{ cm}^2$$

$$f_d = 4,0 \text{ MPa}$$

$$N_{Rdc} = b A_b f_d = 480,0 \text{ kN}$$

$$N_{Edc} = 391,112 \text{ kN}$$

$$N_{Edc} / N_{Rdc} = 0,815 < 1,0 \text{ OK}$$

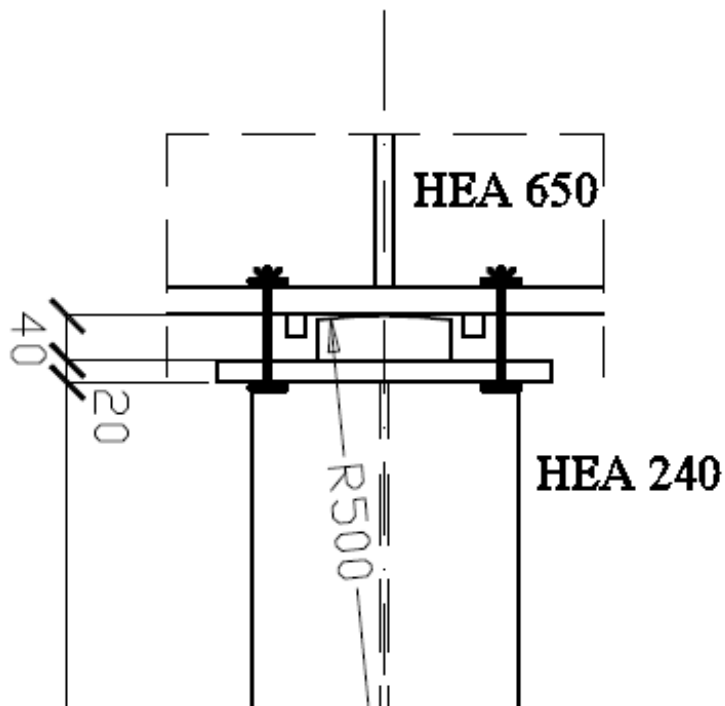
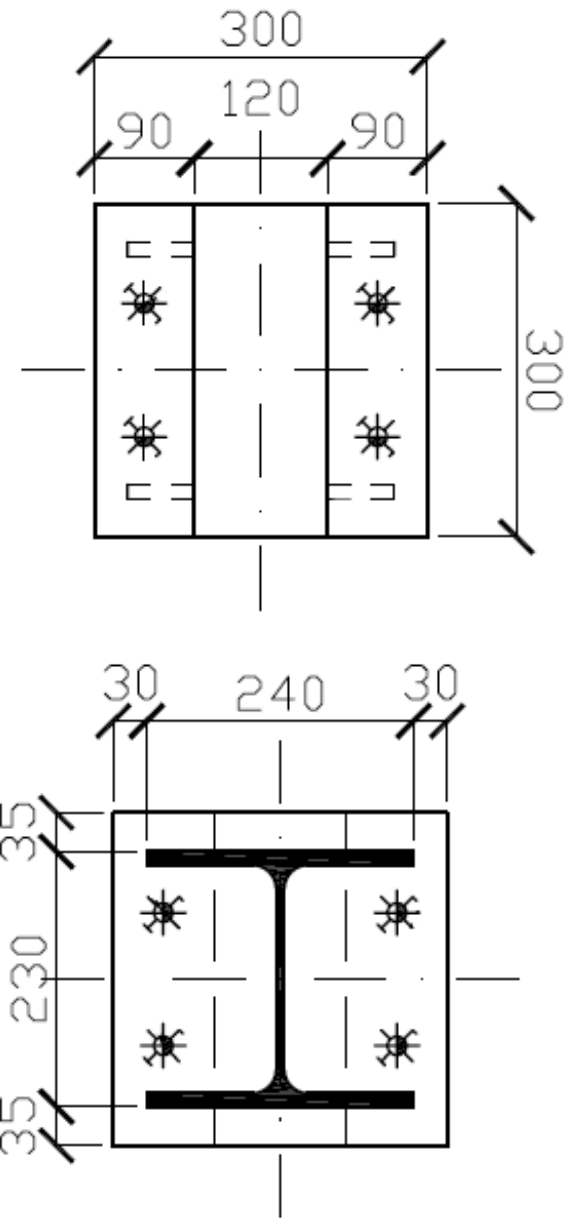


Photo: Author



Geometry of rocker on column's heads is a little another, but general point - round rocker 40 mm thickness, 500 mm radius - is the same in both situations.

Resistance of head of column below support

Bottom plate of rocker in **supported** on three parts of column: two flanges and web.

Along cross-section, marked by red line, bottom plate can be treated as short cantilever (span l_1), loaded by constant load = stresses between rocker and bottom plate

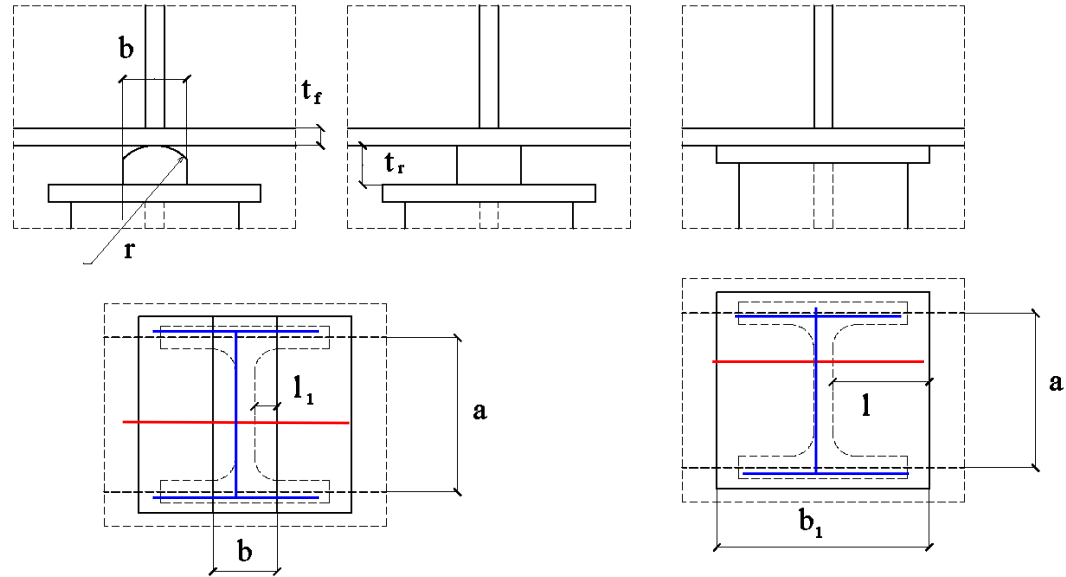


Photo: Author

HEA 240:

$$h = 230 \text{ mm}$$

$$t_f = 12 \text{ mm}$$

$$t_w = 7,5 \text{ mm}$$

$$l_1 = (120 \text{ mm} - 7,5 \text{ mm}) / 2 = 56 \text{ mm}$$

$$a = 230 \text{ mm} - 2 (12 / 2) = 218 \text{ mm}$$

$$q = \sigma = N_{Ed} / (L b) = 1\,184,235 \text{ kN} / (0,3 \text{ m} \cdot 0,12 \text{ m}) = 32\,895,417 \text{ kN} / \text{m}^2$$

$$M_{Ed, \max} (\text{cantilever}) = q a (l_1)^2 / 2 = 11,244 \text{ kNm}$$

Cross-section of short cantilever
(along red line);

thickness t_p ;

width $a = 0,218$ m;

$$W = a t_{p1}^2 / 6$$

$$M_{Rd} = W f_y / \gamma_{M0}$$

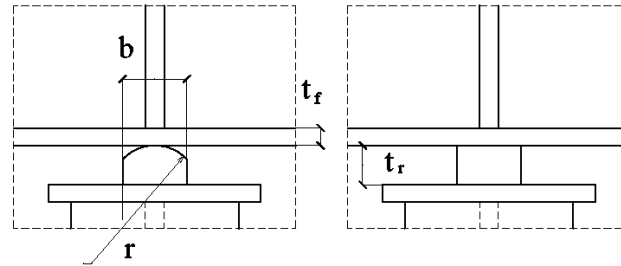
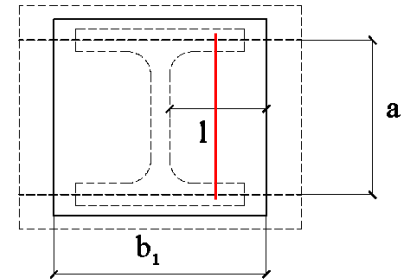
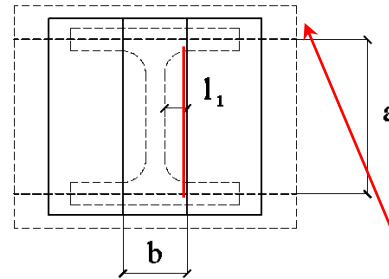
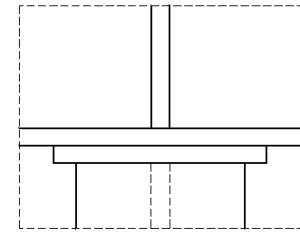


Photo: Author



$$M_{Ed, \max} / M_{Rd} \leq 1,0 \rightarrow M_{Ed, \max} / [f_y a t_{p1}^2 / (6 \gamma_{M0})] \leq 1,0 \rightarrow$$

$$\rightarrow 6 \gamma_{M0} M_{Ed, \max} / (f_y a t_{p1}^2) \leq 1,0 \rightarrow$$

$$\rightarrow t_{p1} \geq \sqrt{[(6 M_{Ed, \max} / (a f_y \gamma_{M0}))]}$$

t_{p1} means: $t_p + t_r$ t_p

Recommended thickness of bottom plate in presence of rocker:

$$t_p \geq \max \{ 0,75 t_f ; \sqrt{[(6 M_{Ed, \max} / (a f_y \gamma_{M0})] - t_r } \}$$

$$t_p \geq \max \{ 0,75 \cdot 26 \text{ mm} ; \sqrt{[(6 \cdot 11,244 \text{ kNm} / (0,218 \text{ m} \cdot 235 \text{ MPa} \cdot 1,0)] - 0,04 \text{ m} } \} =$$

$$= \max (20 \text{ mm} ; 36 \text{ mm} - 40 \text{ mm}) = \max (20 \text{ mm} ; -4 \text{ mm}) = 20 \text{ mm}$$

This thickness of bottom plate will be applied in both cases; rocker on wall and rocker on column.

Cap plate - column weldings

Lecture #17, Example 3a
for
 $M_{Ed} = 0$; $V_{Ed} = 0$

$$\sqrt{[(\sigma_{\perp})^2 + 3(\tau_{\parallel}^2 + \tau_{\perp}^2)]} \leq f_u / (\beta_w \gamma_{M2})$$

$$\sigma_{\perp} \leq 0,9f_u / \gamma_{M2}$$

$$\gamma_{M2} = 1,25$$

$$N_{Ed} = 1\,184,235 \text{ kN}$$

$$f_u = S235 = 360 \text{ MPa}$$

$$\beta_w = S235 = 0,8$$

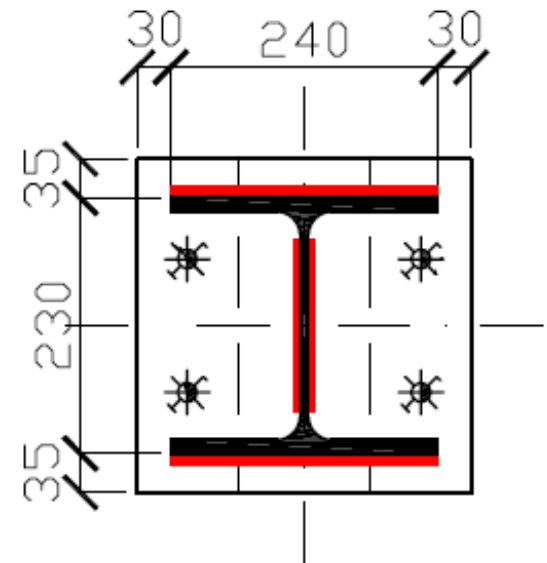


Photo: Author

Cap plate - flange:

$t_1 = \text{thickness of flange} = 12 \text{ mm}$

$t_2 = \text{thickness of cap plate} = 26 \text{ mm}$

$0,2 t_2 \leq a_f \leq 0,7 t_1 \rightarrow a_f = 8 \text{ mm}$

$l_f = \text{length of weld} = 240 \text{ mm}$

Cap plate - web:

$t_1 = \text{thickness of web} = 7,5 \text{ mm}$

$t_2 = \text{thickness of cap plate} = 26 \text{ mm}$

$0,2 t_2 \leq a_w \leq 0,7 t_1 \rightarrow a_w = 5 \text{ mm}$

$l_w = \text{length of weld} = 160 \text{ mm}$

$$A = 2 a_f l_f + 2 a_f l_f = 54,4 \text{ cm}^2$$

$$\sigma (N_{Ed}) = N_{Ed} / A = 217,690 \text{ MPa}$$

$$\sigma_{\perp} = \tau_{\perp} = \sigma / \sqrt{2} = 153,930 \text{ MPa}$$

$$\tau_{\parallel} = 0,000 \text{ MPa}$$

$$\sqrt{[(\sigma_{\perp})^2 + 3(\tau_{\parallel}^2 + \tau_{\perp}^2)]} = 307,860 \text{ MPa}$$

$$f_u / (\beta_w \gamma_{M2}) = 360,0 \text{ MPa}$$

$$307,860 / 360,0 = 0,835$$

condition satisfied

$$\sigma_{\perp} = 153,930 \text{ MPa}$$

$$0,9 f_u / \gamma_{M2} = 259,200 \text{ MPa}$$

$$153,930 / 259,200 = 0,594$$

condition satisfied

Cap plate-column compression

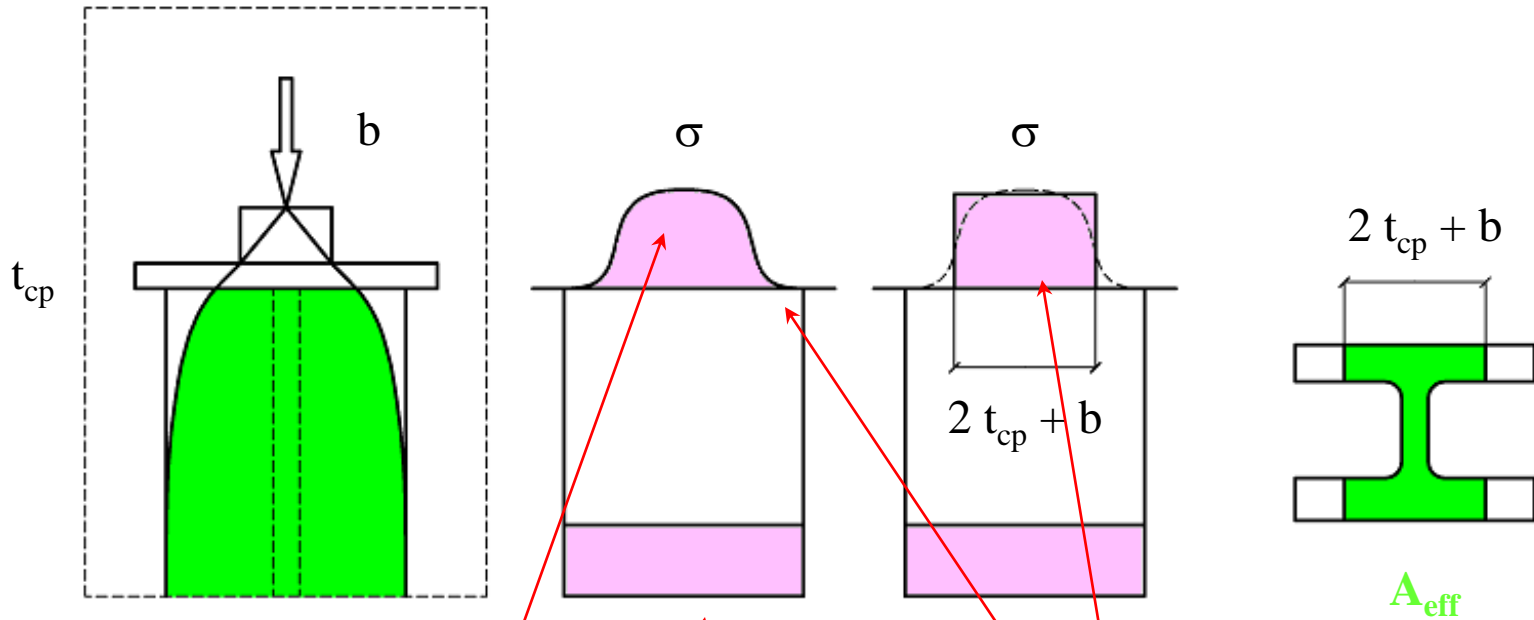


Photo: Author

Cap plate - wide thin member; stress between cap plate and column has non-linear character; we assume big constant value in central part and zero at ends of flange. Constant value of stress exists in distance from end of column. Linearisation of top stress will be applied

$$N_{Ed} / (A_{eff} f_y \gamma_{M0}) \leq 1,0$$

$b = \text{width of rocker} = 120 \text{ mm}$

$t_{cp} = \text{thickness of cup plate} = 20 \text{ mm}$

$b_f = \text{width of column's flange} = 240 \text{ mm}$

$t_f = \text{thickness of column's flange} = 12 \text{ mm}$

$A_{\text{HEA240}} = 76,84 \text{ cm}^2$

$N_{\text{Ed}} = 1055 \text{ kN}$

$f_y = \text{S235} = 235 \text{ MPa}$

$s = b + 2 t_{cp} = 160 \text{ mm}$

$x = b_f - s = 240 - 160 \text{ mm} = 80 \text{ mm}$

$A_{\text{eff}} = A_{\text{HEA240}} - 2 \cdot (x / 2) \cdot t_f =$
 $= 76,84 - 8 \cdot 1,2 = 67,24 \text{ cm}^2$

$A_{\text{eff}} f_y \gamma_{M0} = 1\,510,345 \text{ kN}$

$1\,184,235 / 1\,510,345 = 0,784 < 1,0 \text{ OK}$

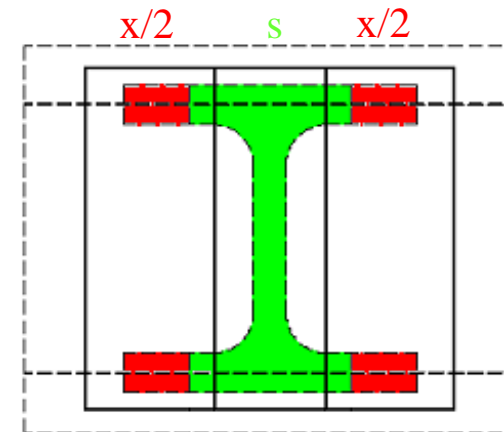
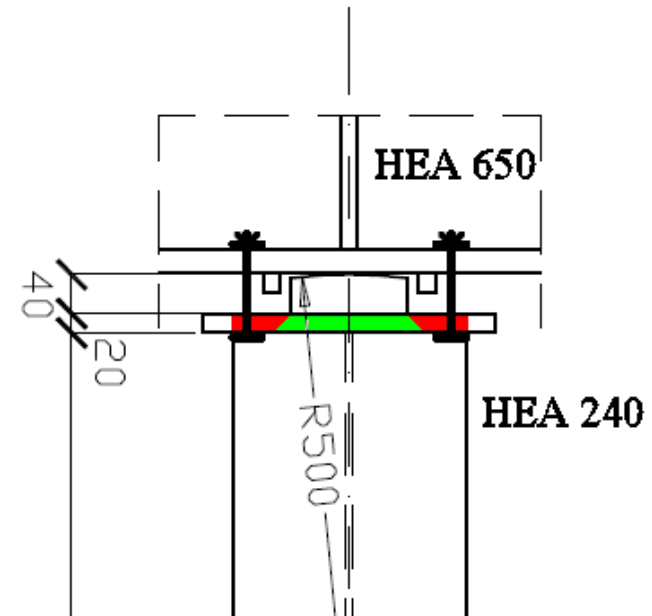


Photo: Author

Vth example of calculations – vertical stiffeners

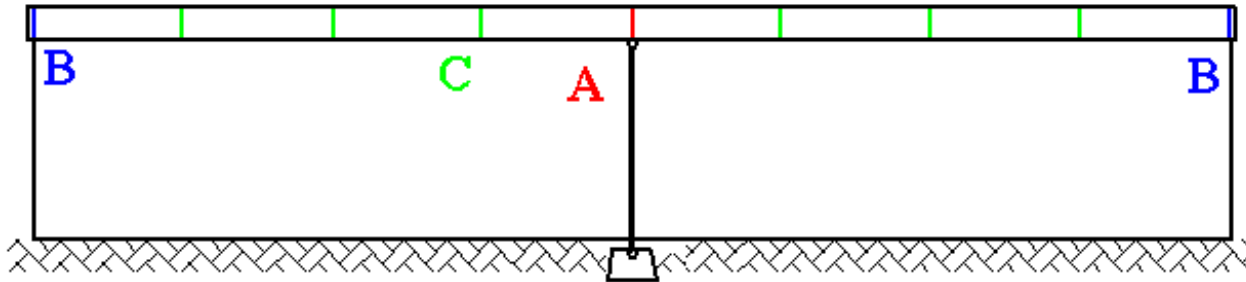


Photo: Author

New complex of data

Secondary beam: IPE 300

S 235 $\rightarrow \varepsilon = 1,0$

Primary beam: HEA 650

A	Reaction from column	1049,9 kN
B	Reaction from wall	287,8 kN
C	2x force from secondary beam	$2 \times 36,4 = 72,8$ kN

Conditions important for vertical stiffeners:

<u>Independent of load</u>	<u>Load-dependent</u>
1. Thickness of adjacent elements (#t / 33)	5. Contact stress (#t / 38)
2. Class of the cross-section (#t / 34)	6. Axial compression (#t / 39-49)
3. Prevent torsional buckling of stiffener (#t / 35)	7. Welds (#t / 50-55)
4. Rigid supports for web panel (#t / 36-37)	

The same value for stiffeners A, B, C:

$$t_{s-in} = \max [(1) ; (2) ; (3) ; (4)]$$

(#t / 35)

Final thickness of stiffener depends on external action, but non smaller than t_{s-in} (#t / 47)

1. Thickness of adjacent elements

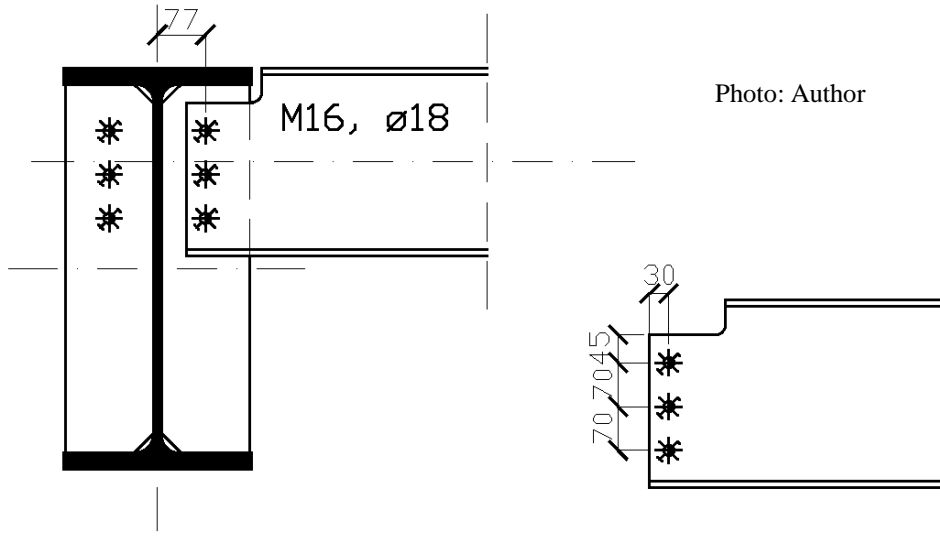
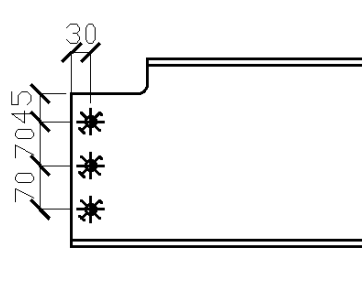


Photo: Author



Vertical stiffener is used to connect secondary beam and primary beam. There are holes for bolts in both members (stiffener / web of secondary beam). Both elements should be analysed because of bearing resistance. Assuming

thickness of stiffener > thickness of secondary beam web

we obtain a higher resistance of stiffener than resistance of web of secondary beam. We don't need to check resistance of stiffener.

$$\underline{t_{s-in, 1} \geq t_{w_{sb}} = 10,7 \text{ mm}}$$

2. Class of the cross-section

2.a. According to old Polish Standard PN-B 3200, stiffeners can't be IVth class of cross-section;

2.b. According to Eurocode EN 1993-1-5, stiffeners are calculated based on total geometry (as for Ist, IInd or IIIrd class of cross-section); not as for IVth where we must use effective geometry.

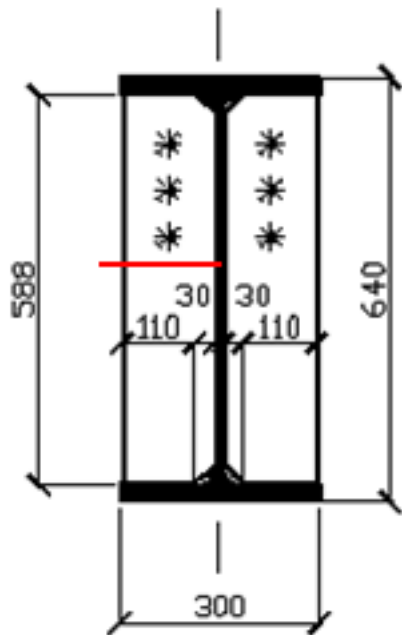


Photo: Author

Stress distribution in parts (compression positive)	
3	$c/t \leq 14\varepsilon$

Photo: EN 1993-1-1 tab. 5.2

Vertical stiffener under compression in horizontal cross-section looks like quasi-flange.

$$b_s / t_s \leq 14 \varepsilon$$

$$b_s = 140 \text{ mm}$$

$$\underline{t_{s-in, 2} \geq 10,0 \text{ mm}}$$

3. Prevent torsional buckling of stiffener

EN 1993-1-5 (9.3)

$$J_T / J_p \geq 5,3 f_y / E$$

$$J_T = b_s t_s^3 / 3$$

$$J_p = b_s t_s (b_s / 2)^2 + b_s t_s^3 / 12 = b_s^3 t_s / 4 + b_s t_s^3 / 12$$

$$5,3 f_y / E = 0,006076$$

$$J_T / J_p = (b_s t_s^3 / 3) / (b_s^3 t_s / 4 + b_s t_s^3 / 12) = b_s t_s (t_s^2 / 3) / [(b_s t_s / 4) (b_s^2 + t_s^2 / 3)] =$$
$$= 4 t_s^2 / (3 b_s^2 + t_s^2)$$

$$4 t_s^2 / (3 b_s^2 + t_s^2) \geq 5,3 f_y / E = 0,006076$$

$$4 t_s^2 \geq 0,006076 (3 b_s^2 + t_s^2)$$

$$t_s^2 (4 - 0,006076) \geq 3 \cdot 0,006076 b_s^2$$

$$t_{s-in,3} \geq 8,1 \text{ mm}$$

Moment of inertia for one stiffener about edge fixed to the plate in horizontal cross-section (J_p) and torsional moment of inertia for one stiffener (J_T).

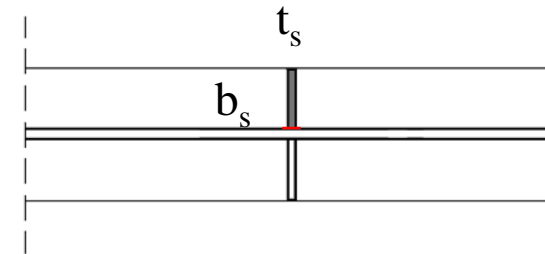


Photo: Author

$$t_{s-in} \geq \max[t_{s-in, 1} ; t_{s-in, 2} ; t_{s-in, 3} ; t_{s-in, 4}] = \max[10,7 \text{ mm} ; 10,0 \text{ mm} ; 8,1 \text{ mm} ; 0,1 \text{ mm}]$$

$$t_{s-in} \geq 10,8 \text{ mm}$$

$$t_{s-in} = 11 \text{ mm}$$

Conditions (1) – (4) are independent of loads; thickness t_{s-in} is the same for each stiffeners.

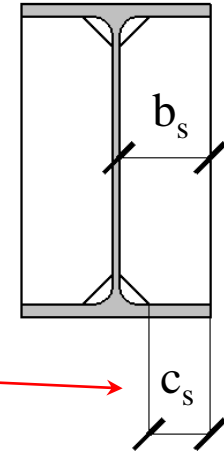
For various stiffeners acts various forces. Is possible, than for big force, fatter stiffener is needed. It will be analysed in second group of conditions: (5), (6), load-dependent.

5. Contact between stiffener and flange of primary beam

EN 1993-1-5 9.4 (2)

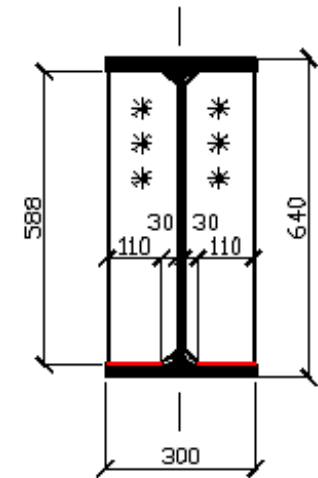
$$F_{s, Ed} / (2 c_s t_s f_y \gamma_{M0}) \leq 1,0 \rightarrow t_s \geq F_{s, Ed} / (2 c_s f_y \gamma_{M0})$$

$$c_s = 110 \text{ mm}$$



	$F_{s, Ed}$	$t_{s, 5}$
A	1049,9 kN	$\geq 20,3 \text{ mm}$
B	287,8 kN	$\geq 5,6 \text{ mm}$
C	$2 \times 36,4 = 72,8 \text{ kN}$	$\geq 1,4 \text{ mm}$

Photo: Author



6. Axial compression

EN 1993-1-5 9.2.1

EN 1993-1-5 9.4 (2)

- ◆ Stiffeners are treated as bar, compressed by axial force $F_{s1, Ed}$;
- ◆ We analyse flexural buckling about x-x axis;
- ◆ We take into consideration cross-section of stiffeners and cooperating part of web (\perp cross-section);
- ◆ Axial force $F_{s1, Ed}$ contains imperfections of stiffeners;
- ◆ Additionally we must analyse imperfections of web - represented by additional load q ;
- ◆ We must analyse interaction between axial force $F_{s1, Ed}$, buckling about x-x and bending moment $M_{s, Ed}$ (q).



Photo: lmsteelfab.com

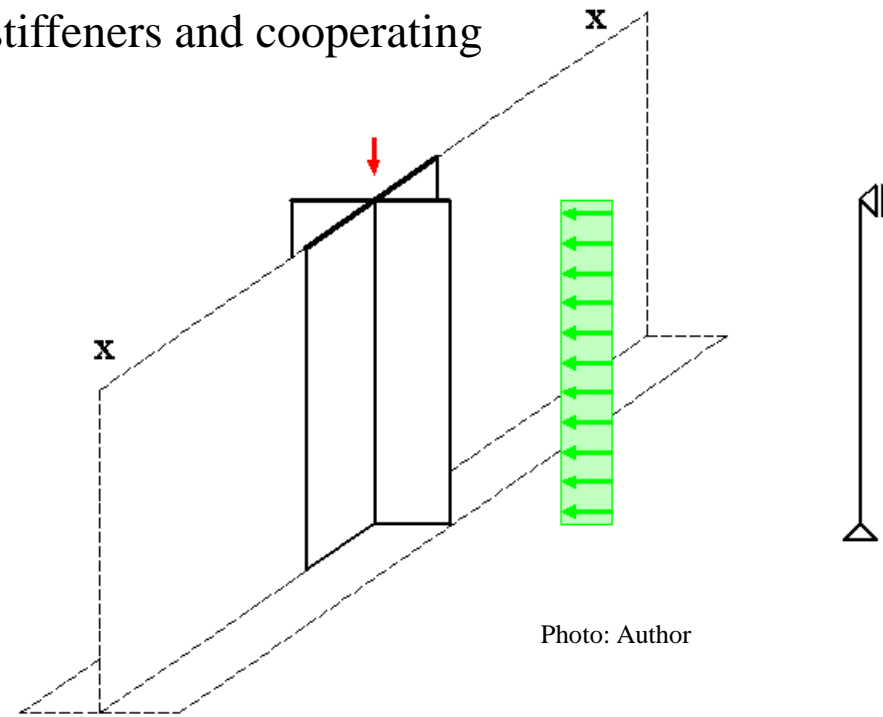


Photo: Author

$$\text{Axial force: } F_{s1, Ed} = \max (F_{s, Ed} + \Delta N_{st} ; V_{Ed}^* + \Delta N_{st})$$

The universal formula, to analyzed different cases:

Analyzed, when force acts directly on the stiffener

Analyzed, when no forces act on the stiffener; stiffener is needed to prevent of web local buckling only.

Design project - violet part only

$$\Delta N_{st} = \sigma_m b^2 / \pi^2$$

$$\sigma_m = (\sigma_{cr, c} / \sigma_{cr, p}) (1 / a_1 + 1 / a_2) N_{Ed, eq} / b$$

$$\sigma_{cr, c} = \pi^2 E t_w^2 / [12 (1 - \nu^2) a^2] \approx 190\,000 (t_w / a)^2 \text{ [MPa]}$$

$$\sigma_{cr, p} = k_s (28,4 \varepsilon)^2 f_y (t_w / b)^2 \approx 190\,000 k_s (t_w / b)^2 \text{ [MPa]}$$

a, a₁, a₂ - distances between secondary beam; regular; irregular

$$a = a_1 = a_2 = 3\,000 \text{ mm}$$

b = h_w - high of primary beam web (HEA 650) = 588 mm

t_w - thickness of primary beam web (HEA 650) = 13,5 mm

N_{Ed, eq} - equivalent axial force in primary beam (→ #t / 41)

ν - Poisson ratio

k_σ - local buckling factor for primary beam (→ # t / 40)

Local buckling factor

EN 1993-1-5, tab 4.1

Stress distribution (compression positive)				Effective ^p width b_{eff}		
				$\psi = 1:$ $b_{eff} = \rho \bar{b}$ $b_{e1} = 0,5 b_{eff} \quad b_{e2} = 0,5 b_{eff}$		
				$1 > \psi \geq 0:$ $b_{eff} = \rho \bar{b}$ $b_{e1} = \frac{2}{5 - \psi} b_{eff} \quad b_{e2} = b_{eff} - b_{e1}$		
				$\psi < 0:$ $b_{eff} = \rho b_c = \rho \bar{b} / (1 - \psi)$ $b_{e1} = 0,4 b_{eff} \quad b_{e2} = 0,6 b_{eff}$		
$\psi = \sigma_2 / \sigma_1$	1	$1 > \psi > 0$	0	$0 > \psi > -1$	-1	$-1 > \psi > -3$
Buckling factor k_σ	4,0	$8,2 / (1,05 + \psi)$	7,81	$7,81 - 6,29\psi + 9,78\psi^2$	23,9	$5,98 (1 - \psi)^2$

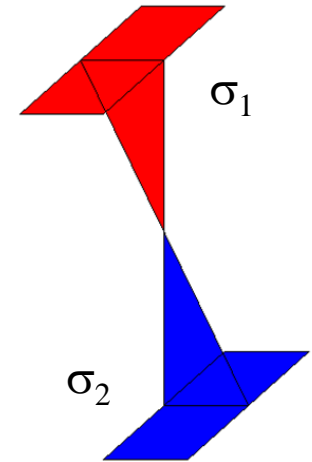


Photo: Author

Global stresses in primary beam web: bending moment only, no axian force

$$\sigma_1 = -\sigma_2 \rightarrow \psi = -1 \rightarrow k_\sigma = 23,9$$

Equivalent axial force

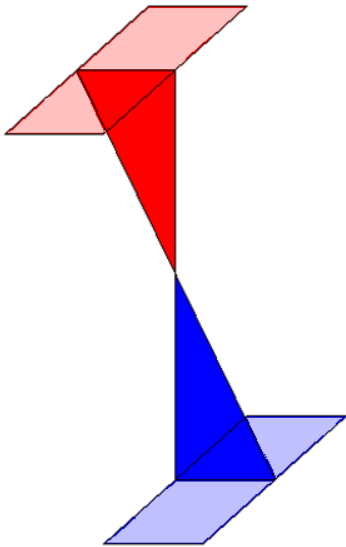


Photo: Author

$N_{Ed, eq}$ – effect of compression in web (**dark red** part only)

For global bending moment only

$$N_{Ed, eq} = \sigma_{max} A / 2$$

A = area of web

Approximation:

$$\sigma_{max} \approx f_y$$

HEA 650:

$$A = 13,5 \text{ mm} \times 588 \text{ mm} = 79,4 \text{ cm}^2$$

$$N_{Ed, eq} = f_y A / 2 = 932,95 \text{ kN}$$

$$a = a_1 = a_2 = 3\,000 \text{ mm}$$

$$h_w = 588 \text{ mm}$$

$$t_w = 13,5 \text{ mm}$$

$$k_s = 23,9$$

$$N_{Ed, eq} = 932,95 \text{ kN}$$

$$\sigma_{cr, c} = \pi^2 E t_w^2 / [12 (1 - \nu^2) a^2] = 190\,000 (t_w / a)^2 [\text{MPa}] = 3,848 \text{ MPa}$$

$$\sigma_{cr, p} = k_s (28,4 \varepsilon)^2 f_y (t_w / b)^2 = 190\,000 k_s (t_w / b)^2 [\text{MPa}] = 2\,393,671 \text{ MPa}$$

$$\sigma_m = (\sigma_{cr, c} / \sigma_{cr, p}) (1 / a_1 + 1 / a_2) N_{Ed, eq} / b = 1,700 \text{ kPa}$$

$$\Delta N_{st} = 0,060 \text{ kN}$$

Additional load q (effect of web imperfections):

$$q = \pi \sigma_m (w_0 + w_{el}) / 4$$

$$M_{s, Ed} = q l^2 / 8$$

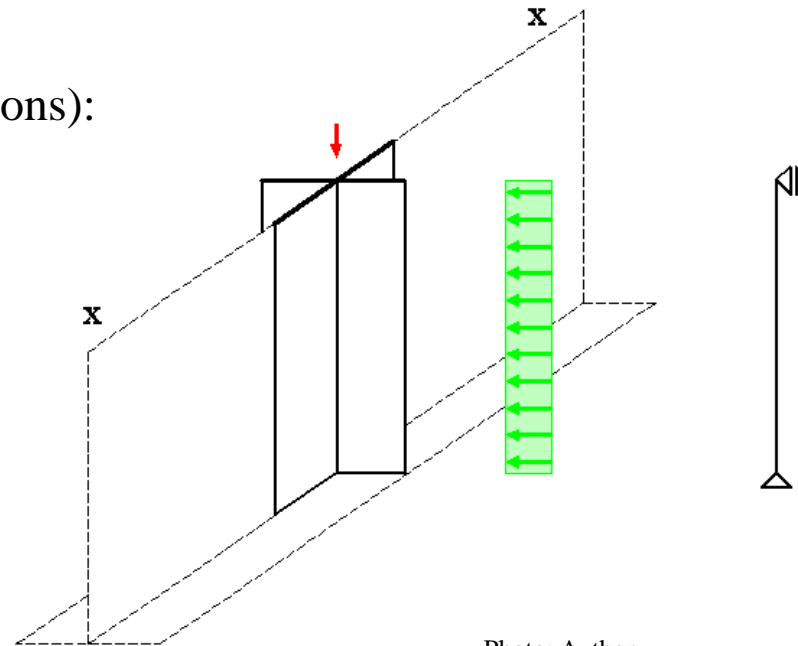


Photo: Author

Full formula for stiffener:

EN 1993-1-1 NA. 20

$$F_{s1, Ed} / (\chi_x N_{Rd}) + M_{s, Ed} / M_{Rd} \leq 1,0 - \Delta_{0, x}$$

Resistance:

$$F_{s1, Ed} = F_{s, Ed} + \Delta N_{st}$$

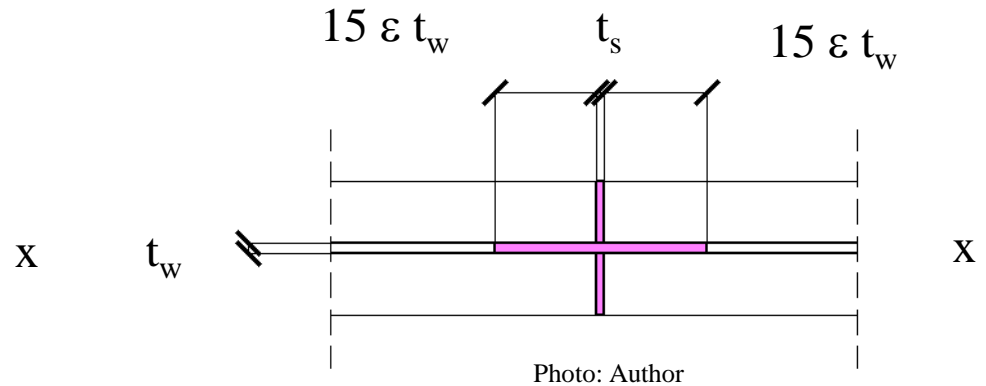
$$M_{s, Ed} = q l^2 / 8$$

$$q = \pi \sigma_m (w_0 + w_{el}) / 4$$

$$w_0 = s / 300$$

$$s = \min (a_1 ; a_2 ; b)$$

$$w_{el} = b / 300$$



N_{Rd} , M_{Rd} and χ_x as for violet cross-section (horizontal cross-section of pair of stiffeners + cooperated part of web).

M_{Rd} and χ_x are calculated for longitudinal axis x-x

Stability:

$$\chi_x = \chi_x (c, \dagger, l_{cr})$$

$$l_{cr} = \mu h_w$$

$$\mu = 0,75$$

$$b = h_w = 588 \text{ mm}$$

$$b_s = 140 \text{ mm}$$

$$a = a_1 = a_2 = 3\,000 \text{ mm}$$

$$\sigma_m = 1,700 \text{ kPa}$$

$$s = \min(a_1 ; a_2 ; b) = 588 \text{ mm}$$

$$w_0 = s / 300 = 1,96 \text{ mm}$$

$$w_{el} = b / 300 = 1,96 \text{ mm}$$

$$q = \pi \sigma_m (w_0 + w_{el}) / 4 = 0,005 \text{ kN / m}$$

$$l = h_w$$

$M_{s, Ed} = q l^2 / 8 = 0,0004 \text{ kNm}$ – negligible, there will be analysed buckling under axial force $F_{s1, Ed}$ only

Stiffener	$F_{s, Ed} + \Delta N_{st}$	t_{s-in}	A_{\dagger}	$N_{Rd} = A_{\dagger} f_y$
A	1049,96 kN	11 mm	$2 \cdot 15 \varepsilon t_w +$ $+ t_{s, 1-4} (2 b_s + t_w) =$ $= 72,785 \text{ cm}^2$	1710,446 kN
B	287,86 kN			
C	72,86 kN			

$$J_{x, \dagger} = (2 b_s + t_w)^3 t_{s, 1-4} / 12 + 2 \cdot 15 \varepsilon (t_w)^4 / 12 \approx (2 b_s + t_w)^3 t_{s, 1-4} / 12 = 2\,317,585 \text{ cm}^4$$

$$i_{x, \dagger} = \sqrt{(J_{x, \dagger} / A_{\dagger})} = 5,643 \text{ cm}$$

$$l_{cr} = h_w \mu = 44,1 \text{ cm}$$

$$\lambda_1 = 93,9 \varepsilon = 93,9$$

$$\lambda = (l_{cr} / i) (1 / \lambda_1) = 0,083$$

$\lambda < 0,2$ – buckling is not important, $\chi = 1,0$ no impact of lost of stability, resistance checked on #t / 36

Stiffener	$F_{s1, Ed}$	χN_{Rd}	E / R
A	1049,96 kN	1710,446 kN	0,614
B	287,86 kN		0,168
C	72,86 kN		0,043

Conclusions:

From **conditions 1-4**, 11 mm must be applied for each stiffeners;

From **condition 5**, $t_{s,A} > 20,3$ mm, for rest 11 mm is enough.

From **condition 6**, 11 mm is enough for each stiffeners;

Final decision:

Stiffener	t_s
A	21 mm
B	11 mm
C	11 mm

7. Welds

Lecture #17, Example 1
for
 $N_{Ed} = 0$

Welds between transverse stiffeners and web



Photo: steelconstruction.info

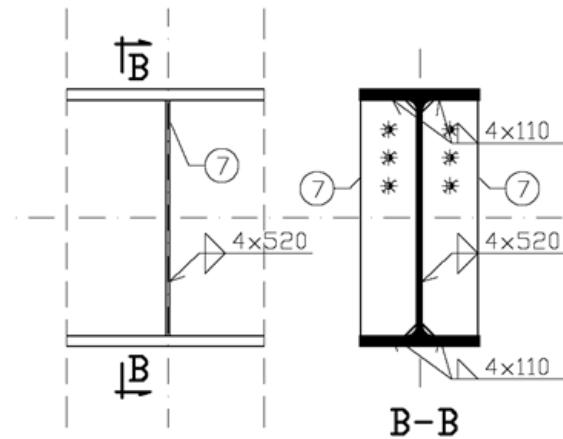


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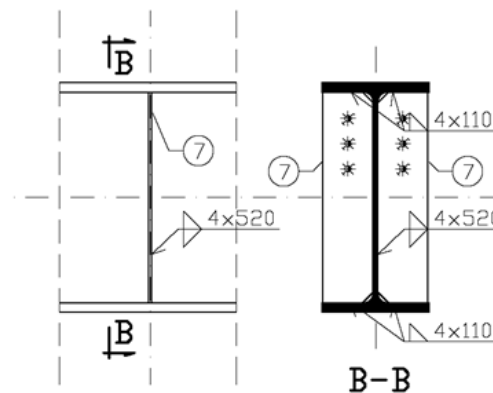
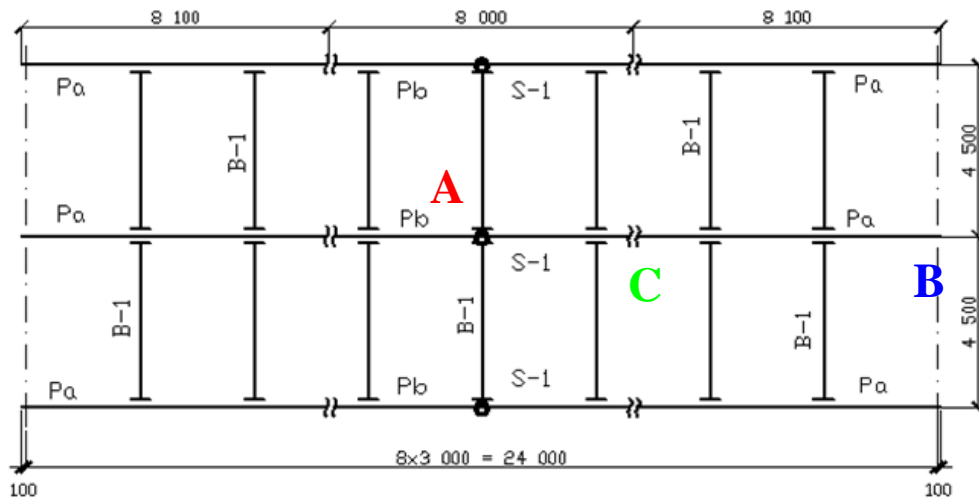


Photo: Author

A	Column - reaction	1049,9 kN
B	Wall - reaction	287,8 kN
C	2x force from secondary beam	2x36,4 = 72,8 kN

Position of center of gravity depends on thickness of welds. Thickness of weld a between two elements of thickness t_1 and t_2 could be taken into consideration according to old Polish Standard:

$$0,2 t_2 \leq a \leq 0,7 t_1$$

$$t_2 \geq t_1$$

(PN-B 3200)

Thickness of stiffeners:

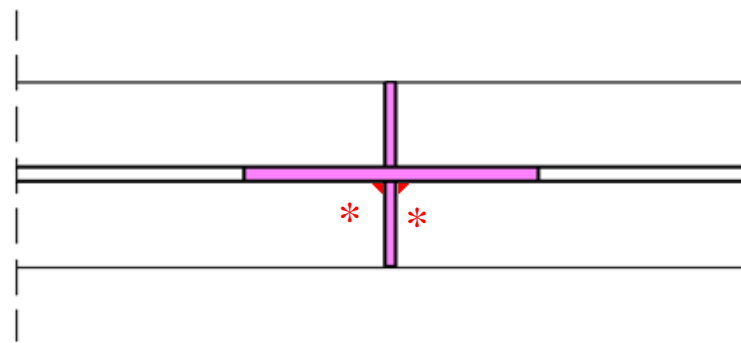
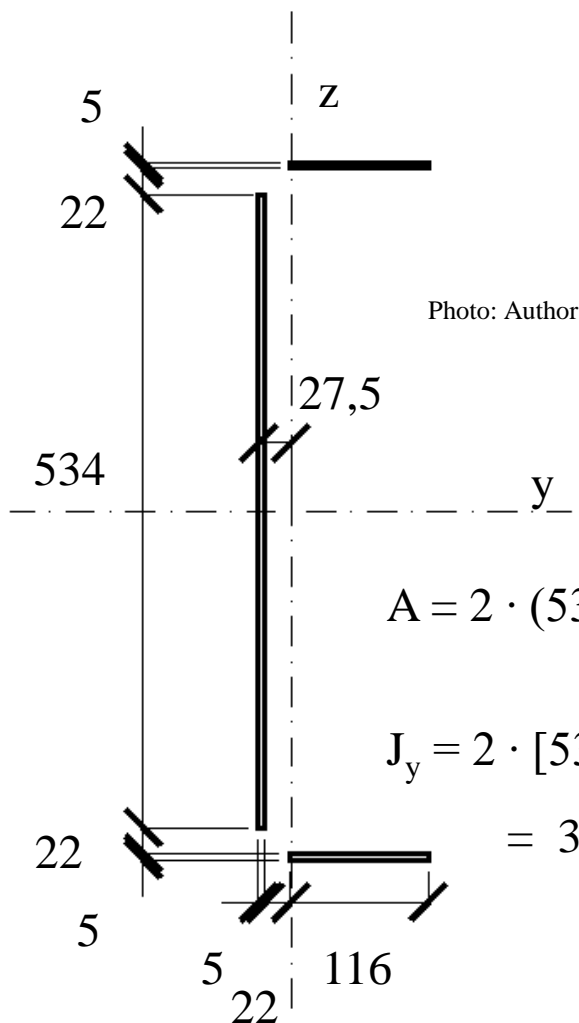
Stiffener	t_s
A	21 mm
B	11 mm
C	11 mm

Thickness of web (HEA 650) = 13,5 mm

Thickness of flange (HEA 650) = 26 mm

Stiffener	Stiffener - flange		Stiffener- web	
	A	$t_2 = 26 \geq t_1 = 21$	$5 \leq a \leq 15$	$t_2 = 21 \geq t_1 = 13,5$
B	$t_2 = 26 \geq t_1 = 11$	$5 \leq a \leq 8$	$t_2 = 13,5 \geq t_1 = 11$	$3 \leq a \leq 8$
C				

For each welds for calculations will be taken into consideration $a = 5$ mm

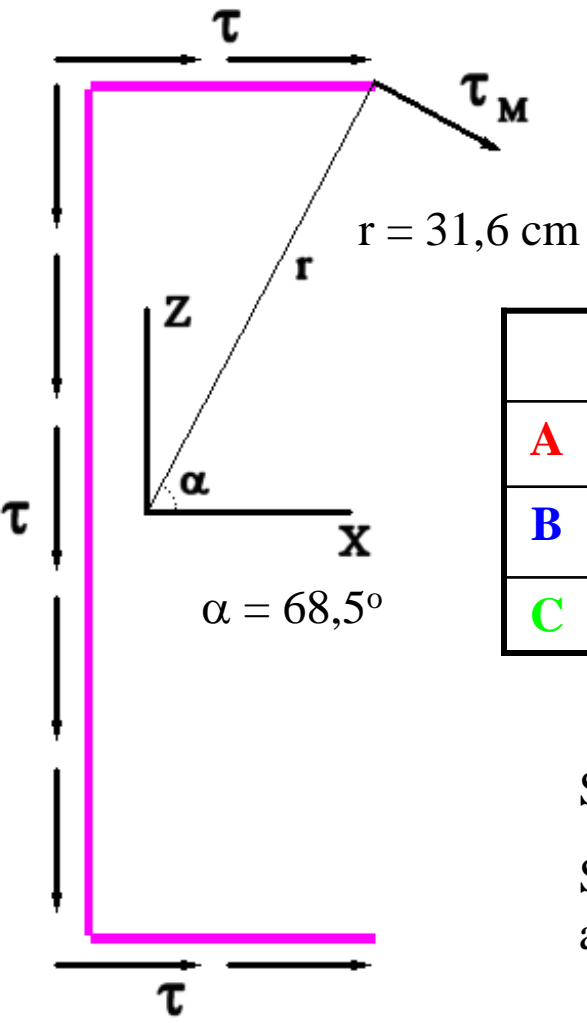


Geometrical characteristics of welds are multiplied by 2 because of 2 times complex of welds (on left and right side of stiffener).

$$A = 2 \cdot (53,4 \cdot 0,5 + 2 \cdot 11,6 \cdot 0,5) = 76,6 \text{ cm}^2$$

$$J_y = 2 \cdot [53,4^3 \cdot 0,5 / 12 + 2 \cdot 11,6 \cdot 0,5 \cdot (53,4 / 2 + 2,2 + 0,5 / 2)^2] = 32\,403 \text{ cm}^4$$

$$J_z = 2 \cdot [2 \cdot 11,6^3 \cdot 0,5 / 12 + 53,4 \cdot 0,5 \cdot (2,75 - 0,5/2)^2 + 2 \cdot 11,6 \cdot 0,5 \cdot (0,5 + 2,2 + 11,6/2 - 2,75)^2] = 1444 \text{ cm}^4$$



$$\tau = V_{Ed} / A$$

$$\bar{\tau}_M = \overline{M r} / J_o$$

$$J_o = J_y + J_z$$

	V_{Ed}	M_{Ed}	τ	$\bar{\tau}_M$
A	524,95 kN	14,44 kNm	68,53 MPa	13,48 MPa
B	143,60 kN	3,95 kNm	18,75 MPa	3,69 MPa
C	36,40 kN	1,00 kNm	4,75 MPa	0,93 MPa

Stress $\tau(V_{Ed})$ is parallel to axis of welds ($= \tau_{||}$)

Stress $\tau(M_{Ed})$ must be recalculated to $\tau_{||}$ and τ_{\perp} by $\sin \alpha$ and $\cos \alpha$

Photo: Author

	$\tau_{\parallel}(V_{Ed})$	$\tau(M_{Ed})$	$\tau_{\parallel}(M_{Ed})$	$\tau_{\perp}(M_{Ed})$	$\tau_{\parallel} = \tau_{\parallel}(V_{Ed}) + \tau_{\parallel}(M_{Ed})$
A	68,53 MPa	13,48 MPa	12,54 MPa	4,94 MPa	81,07 MPa
B	18,75 MPa	3,69 MPa	3,43 Mpa	1,35 MPa	22,18 MPa
C	4,75 MPa	0,93 MPa	0,87 MPa	0,34 MPa	5,62 MPa

The biggest stresses are for stiffener A. This is enough to calculate weld for this one case.

$$\sqrt{[(\sigma_{\perp})^2 + 3(\tau_{\parallel}^2 + \tau_{\perp}^2)]} = 140,678 \text{ MPa} < f_u / (\beta_w \gamma_{M2}) = 360 \text{ MPa} \text{ OK}$$

$$\sigma_{\perp} = 0,000 \text{ MPa} < 0,9f_u / \gamma_{M2} = 259,200 \text{ MPa} \text{ OK}$$

VIth example of calculations - column base

New complex of data:

$$N_{Ed} = 1\,050,0 \text{ kN}$$

Column HEA 240

S235

Concrete base C 30/37 $f_{ck} = 30 \text{ MPa}$



Photo: j-p.com.ua

Analysis of supporting joint concerns few important questions:

- Stiffness of joint (according assumption – hinge joint) → #t / 60
- Resistance of concrete onder base plate → #t / 61 - 62
- Welds column-base plate → #t / 63

One of two methods of calculation can be appicated:
rough or accurate method.

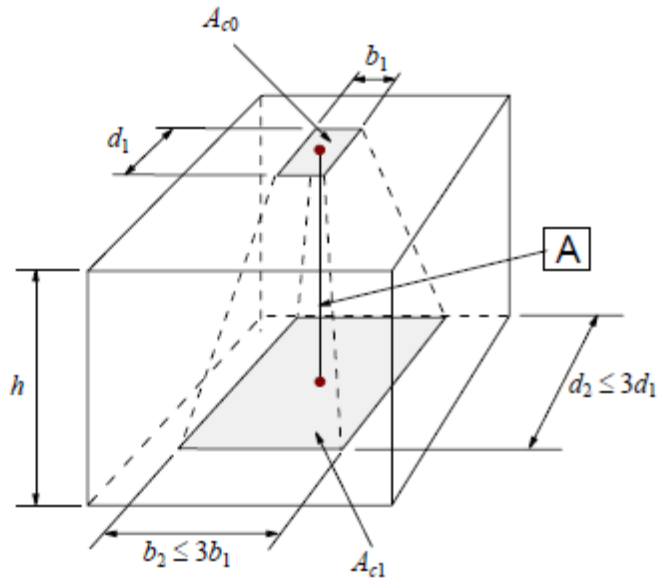


Photo: EN 1992-1-1 fig 6.29

$$h \geq (b_2 - b_1) \text{ i } h \geq (d_2 - d_1)$$

Accurate method based on analysis of stress
distribution in concrete base block.

Rough analysis based on one fundamental assumption: horizontal area of concrete
base block can't be smaller than 2,25 area of steel base plate.

Rough analysis will be adopted into calculation.

Both methods are based on observation: distribution of stresses from steel base plate against concrete foundation is non-linear. The highest values occur under column shaft and in immediate vicinity, while they decrease further away. Linearization is assumed: a large constant stress value under shaft and in its vicinity, zero value in remaining area. Vicinity is described by distance c , depending on thickness of base plate t_p . Thickness of base plate t_p should be calculated for condition: stresses can't exceed strength of concrete.

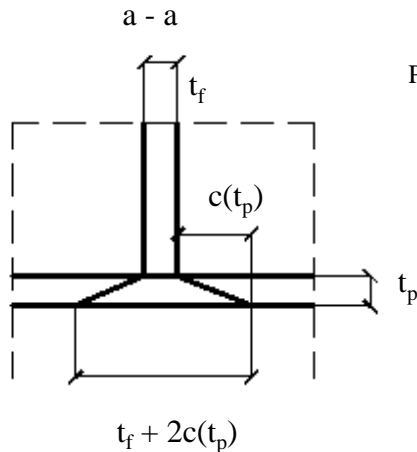


Photo: Author

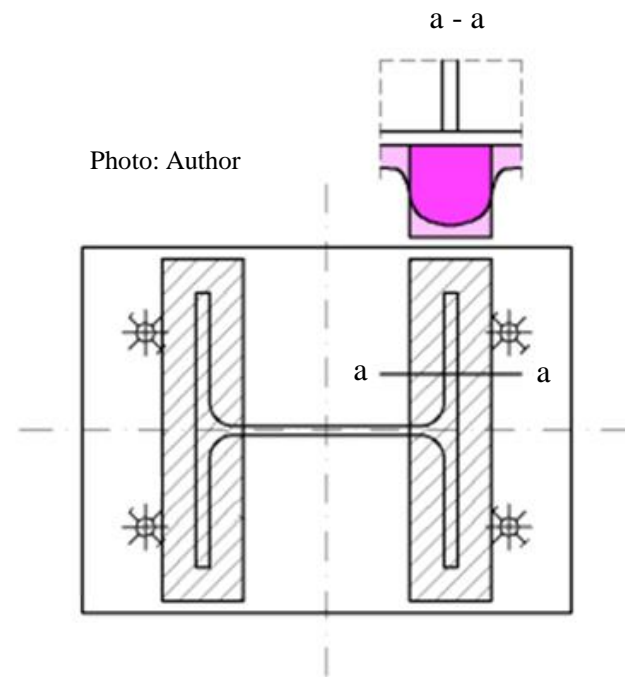
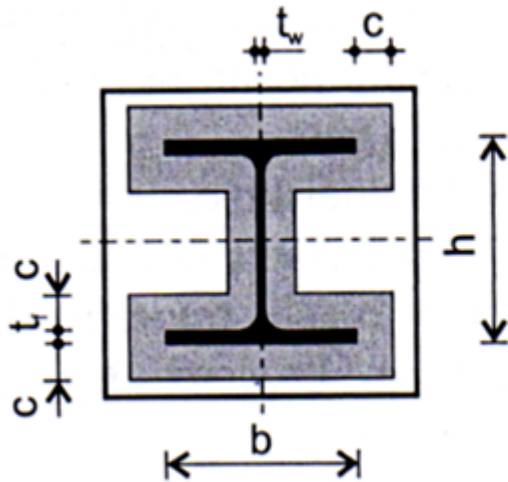


Photo: Author

Stiffness of column's base

According to #15 / 79, column's base with one couple of anchor bolts, located in strong axis of column's cross-section is hinge support. This means, that real behaviour of support is the same as in initial assumption (hinge support).

Resistance of concrete block



Nośność podstawy słupa:

$$N_{Rd} = f_{cd} \left[2(b + 2c)(t_f + 2c) + (t_w + 2c)(h - 2c - 2t_f) \right]$$

Maksymalny wysięg strefy docisku:

$$c = - \frac{X_2 - \sqrt{X_2^2 - 4X_1X_3 + 4X_1N_{Rd}}}{2X_1} *$$

$$X_1 = 4f_{cd}$$

$$X_2 = 4bf_{cd} + 2hf_{cd} - 2t_w f_{cd}$$

$$X_3 = 2bt_f f_{cd} + ht_w f_{cd} - 2t_f t_w f_{cd}$$

There is an error in source, it should be N_{Ed} , not N_{Rd}

c and N_{Rd} according to table:

"Konstrukcje stalowe, przykłady obliczeń według PN-EN 1993-1, część II, stropy i pomosty", praca zbiorowa pod redakcją A. Kozłowskiego, Oficyna Wydawnicza Politechniki Rzeszowskiej, Rzeszów 2009.

$$t_f (\text{HEA 240}) = 12 \text{ mm}$$

$$t_w (\text{HEA 240}) = 7,5 \text{ mm}$$

$$h (\text{HEA 240}) = 230 \text{ mm}$$

$$b (\text{HEA 240}) = 240 \text{ mm}$$

$$f_{cd} = \alpha_{cc} f_{ck} / \gamma_c = 30 \text{ MPa} \cdot 1 / 1,5 = 20 \text{ MPa}$$

$$X_1 = 4 f_{cd} = 80 \text{ MPa}$$

$$X_2 = 4 b f_{cd} + 2 h f_{cd} - 2 t_w f_{cd} = 28,1 \text{ MN/m}$$

$$X_3 = 2 b t_f f_{cd} + 2 h t_w f_{cd} - 2 t_w t_f f_{cd} = 0,181 \text{ MN}$$

$$N_{Ed} = 1,050 \text{ MN}$$

$$c = -\{X_2 - \sqrt{[X_2^2 - 4 X_1 X_3 + 4 X_1 N_{Ed}]}\} / (2 X_1) = 29 \text{ mm}$$

$$\text{Thickness of base plate } t_p = c / \sqrt{[f_y / (3 f_{jd} \gamma_{M0})]} = 15 \text{ mm}$$

$$N_{Rd} = f_{cd} [2(b + 2c)(t_f + 2c) + (t_w + 2c)(h - 2c - 2 t_f)] = 1087,443 \text{ kN}$$

$$1050 \text{ kN} / 1087,443 \text{ kN} = 0,966 < 1,0 \text{ OK}$$

Base plate dimensions: non smaller than $(c + b + c) \times (c + h + c) =$
 $= (29 + 240 + 29) \times (29 + 230 + 29) = 298 \times 288$; it could be taken as 300 mm x 290 mm.

Welds

Welds between shaft of column and base plate can be taken the same as between column and bottom plate of rocker (#t / 26 – 28)

Thank you for attention

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